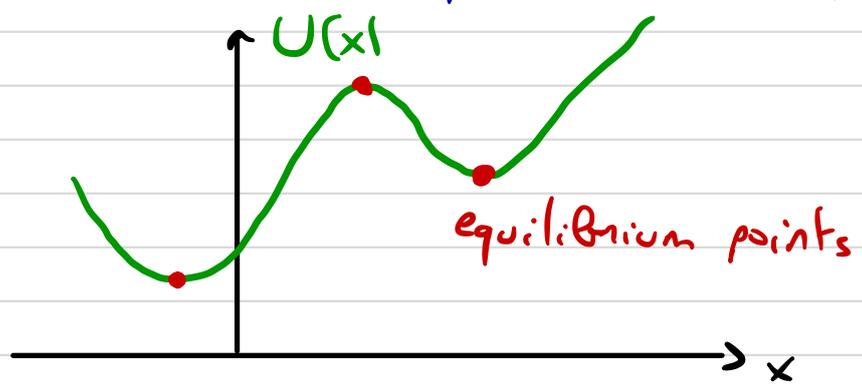


# Small Oscillations

## I Single particle

### a) Equilibrium, stability and small oscillations

• Focus on 1d system for simplicity, conservative



• Equilibrium points:

$$\left. \frac{dU}{dx} \right|_{x_*} = 0$$

$$(F(x_*) = 0)$$

No motion if we start from these points.

• What about if we go slightly away?

$$x = x_* + \delta x \quad \text{with} \quad \delta x \equiv x - x_* \quad \text{small}$$

Taylor expand:  $\rightarrow$  cst

$$U(x) = U(x_*) + \delta x \cancel{U'(x_*)} + \frac{\delta x^2}{2} U''(x_*) + \dots$$

$$\approx \frac{\delta x^2}{2} U''(x_*) + \text{cst}$$

0 by definition of  $x_*$

$$F = - \frac{dU}{dx} = - \frac{dU}{d\delta x} = - U''(x_*) \delta x \quad \rightarrow \text{looks like a spring if } U''(x_*) > 0$$

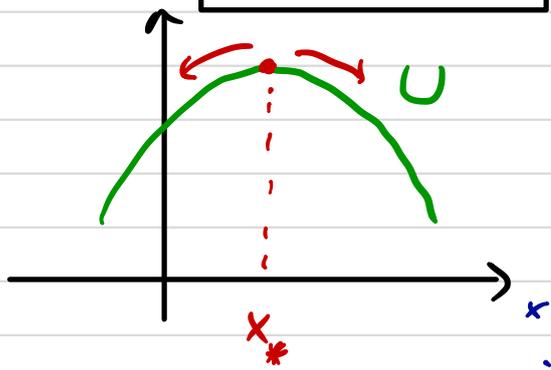
Equation of motion:

$$m \delta \ddot{x} + U''(x_*) \delta x = 0$$

We need to find two independent solutions. Look for exponentials:

try  $e^{\alpha t}$  :  $m \alpha^2 + U''(x_*) = 0$   
 $\Rightarrow \alpha = \pm \sqrt{\frac{-U''(x_*)}{m}}$

so if  $U''(x_*) < 0$  :



$$\alpha = \pm \sqrt{\frac{|U''(x_*)|}{m}} = \pm \alpha_*$$

$$\delta x = A e^{-\alpha_* t} + B e^{\alpha_* t}$$

initial conditions:  $\delta x_0 = A + B$

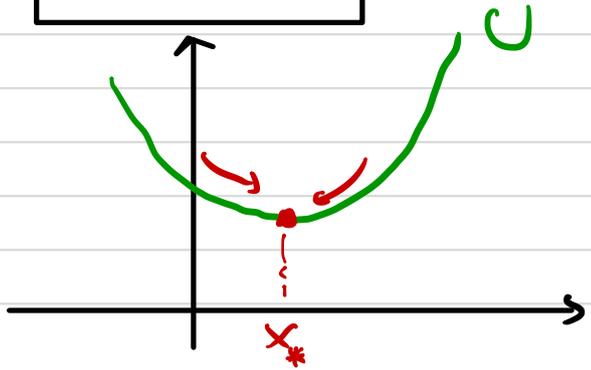
$$\delta \dot{x}(t=0) = 0 = -\alpha_* A + B \alpha_* \Rightarrow A = B$$

$$\Rightarrow \delta x(t) = \frac{\delta x_0}{2} (e^{\alpha_* t} + e^{-\alpha_* t}) = \delta x_0 \cosh(\alpha_* t)$$

$\Rightarrow$  diverges as  $t \rightarrow \infty$ : assumption of small  $\delta x$  breaks down!

$\Rightarrow$  UNSTABLE (moves away from  $x_*$ )

if  $U''(x_*) > 0$



$$\alpha = \pm i \sqrt{\frac{U''(x_*)}{m}} = \pm i \omega_0$$

$$\omega_0^2 = \frac{U''(x_*)}{m} \quad (\delta \ddot{x} + \omega_0^2 \delta x = 0)$$

$\Rightarrow e^{\pm i\omega_0 t}$  solutions or equivalently  $\cos(\omega_0 t)$  and  $\sin(\omega_0 t)$

$$\Rightarrow \delta x(t) = \delta x_0 \cos(\omega_0 t) \quad \left( \text{Period } T = 2\pi \sqrt{\frac{m}{U''(x_*)}} \right)$$

$\Rightarrow$  **STABLE**. Oscillation around  $x_*$ .

Approach valid for  $\delta x_0$  small, otherwise anharmonic terms  $\delta x^3, \delta x^4, \dots$  become important.

## b) Damped Oscillations

• Add a drag / damping force:  $\vec{F}_d = -\gamma \dot{x} \vec{e}_x = -\gamma \vec{v}$

$\rightarrow$  non conservative, expect energy loss and damping of oscillations

$$m \ddot{x} = -\gamma \dot{x} - kx \quad \Rightarrow \quad \boxed{\ddot{x} + 2\beta \dot{x} + \omega_0^2 x = 0}$$

$\hookrightarrow \beta = \gamma/2m$

• Look for  $e^{\alpha t}$  solutions:  $\alpha^2 + 2\beta\alpha + \omega_0^2 = 0$

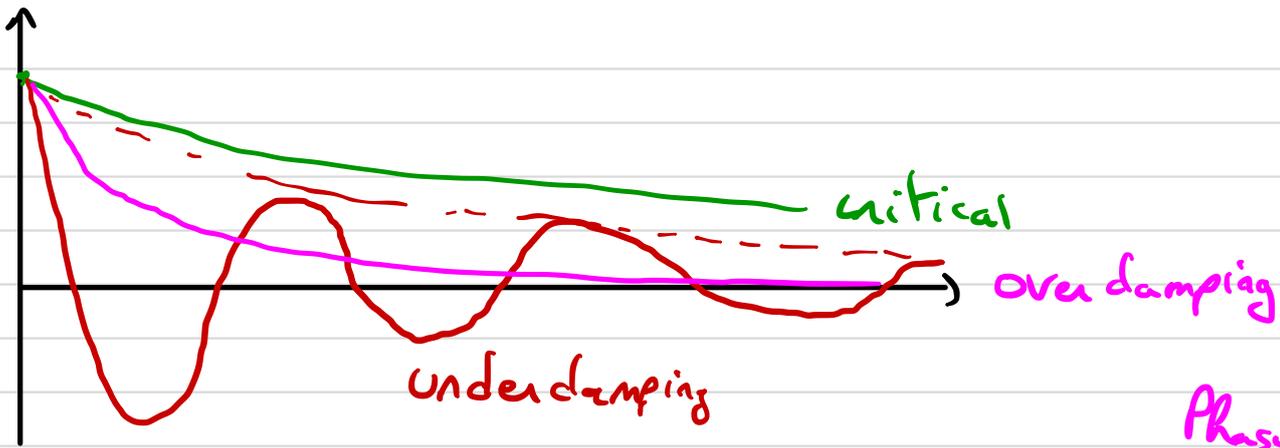
$$\alpha_{\pm} = -\beta \pm \sqrt{\beta^2 - \omega_0^2}$$

General solution:

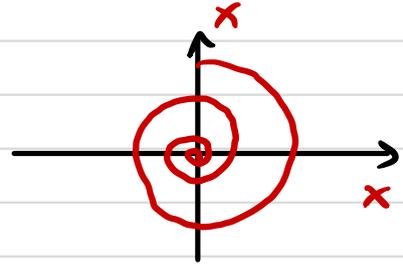
$$x(t) = e^{-\beta t} \left[ A e^{\sqrt{\beta^2 - \omega_0^2} t} + B e^{-\sqrt{\beta^2 - \omega_0^2} t} \right]$$

**Three Cases:**

- $\omega_0^2 > \beta^2$ : underdamped motion
- $\beta^2 > \omega_0^2$ : overdamped motion
- $\beta^2 = \omega_0^2$ : critical case



Phase portrait:



$\omega_0^2 > \beta^2$ :  $\alpha_{\pm} = -\beta \pm i\sqrt{\omega_0^2 - \beta^2}$

$x(t) = e^{-\beta t} [A e^{i\omega_d t} + B e^{-i\omega_d t}]$

$= C e^{-\beta t} \cos(\omega_d t + \phi)$   $\omega_d = \sqrt{\omega_0^2 - \beta^2}$

→ damped oscillations

$\omega_0^2 < \beta^2$ :  $x(t) = e^{-\beta t} [A e^{-\omega' t} + B e^{+\omega' t}]$

$\omega' = \sqrt{\beta^2 - \omega_0^2} < \beta$  : decaying exponentials

$\omega_0^2 = \beta^2$  :  $x(t) = (A + Bt) e^{-\beta t}$

↑ additional solution

c) Periodic driving forces

ex:  $F = -Kx - \gamma \dot{x} + F_0 \cos \omega t$

$$\beta = \frac{\gamma}{2m}, \omega_0^2 = \frac{k}{m}, A = F_0/m \quad (5)$$

$$\Rightarrow m\ddot{x} = F_0 \cos \omega t - kx - \alpha \dot{x} \Rightarrow \boxed{\ddot{x} + 2\beta \dot{x} + \omega_0^2 x = A \cos \omega t}$$

Solution:  $x(t) = x_c(t) + x_p(t)$  ↗ particular solution

↙ complementary solution: RHS = 0  
("homogeneous eq")

$$x_c = e^{-\beta t} \left[ A_1 e^{\sqrt{\beta^2 - \omega_0^2} t} + A_2 e^{-\sqrt{\beta^2 - \omega_0^2} t} \right]$$

(see previous section)

↪ describes "transient"  
(decay to 0 as  $t \gg \beta^{-1}$ )

Particular solution: try  $x_p = D \cos(\omega t - \delta)$

↪ same frequency as the drive (Linear eq!)

Trick: write  $x_p = \text{Re} [ \underline{D} e^{i\omega t} ]$ ,  $\underline{D} = D e^{-i\delta} \in \mathbb{C}$

$$A \cos \omega t = \text{Re} [ A e^{i\omega t} ]$$

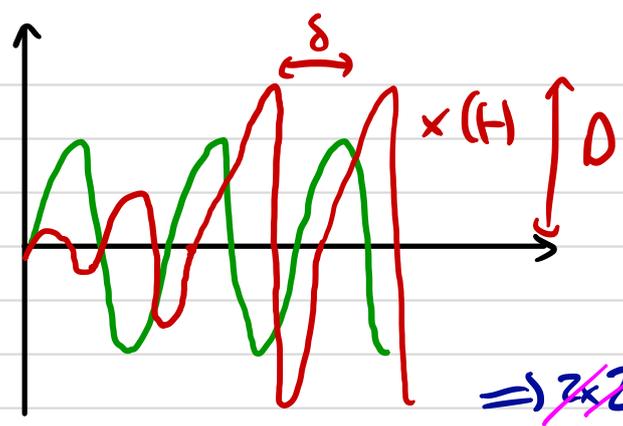
$$\Rightarrow [-\omega^2 + 2\beta i\omega + \omega_0^2] \underline{D} e^{i\omega t} = A e^{i\omega t}$$

$$\Rightarrow \underline{D} = \frac{A}{\omega_0^2 - \omega^2 + i2\beta\omega} = \frac{(\omega_0^2 - \omega^2)A - i2\beta\omega A}{(\omega_0^2 - \omega^2)^2 + 4\beta^2\omega^2}$$

$$\boxed{D = |\underline{D}| = \frac{A}{\sqrt{(\omega^2 - \omega_0^2)^2 + 4\beta^2\omega^2}}, \quad \tan \delta = \frac{2\beta\omega}{\omega_0^2 - \omega^2}}$$

↪ "delay", phase difference

$$x(t \gg \beta^{-1}) \approx x_p(t) \quad (\text{steady-state solution})$$



Amplitude resonance  
frequency:

$$\frac{dD}{d\omega} \Big|_{\omega_R} = 0$$

$$\Rightarrow 2\omega_R (\omega_R^2 - \omega_0^2) + 2\beta^2 \omega_R = 0$$

$$\Rightarrow \omega_R = \sqrt{\omega_0^2 - 2\beta^2} \quad (\text{if } \beta > \omega_0/\sqrt{2} : \text{no resonance})$$

D decreases monotonically  $\omega/\omega_0$

Quality factor :  $Q \equiv \frac{\omega_R}{2\beta}$  ( $Q \rightarrow \infty$  as  $\beta \rightarrow 0$  no damping)

Principle of superposition

$$\left( \frac{d^2}{dt^2} + 2\beta \frac{d}{dt} + \omega_0^2 \right) x = A \cos \omega t = F(t)/m$$

$\hat{L}$  = Linear operator :  $\hat{L}(x_1 + x_2) = \hat{L}x_1 + \hat{L}x_2$

if  $\hat{L}x_1 = F_1$   
 $\hat{L}x_2 = F_2$   $\Rightarrow \hat{L}(\alpha_1 x_1 + \alpha_2 x_2) = \alpha_1 F_1 + \alpha_2 F_2$

So if  $F(t) = \sum_n \alpha_n \cos(\omega_n t - \phi_n)$  (RHS)

The steady-state solution is:

$$x_p(t) = \frac{1}{m} \sum_n \frac{\alpha_n}{\sqrt{(\omega_0^2 - \omega_n^2)^2 + 4\omega_n^2 \beta^2}} \cos(\omega_n t - \phi_n - \delta_n)$$

$\hookrightarrow \tan^{-1} \left( \frac{2\omega_n \beta}{\omega_0^2 - \omega_n^2} \right)$

So we know the solution for **any** periodic force  $F(t) = F(t+T)$   
 $T = 2\pi/\omega$

Thanks to **Fourier Series**:

$$F(t) = \frac{1}{2} a_0 + \sum_{n=1}^{\infty} (a_n \cos n\omega t + b_n \sin n\omega t)$$

$$a_n = \frac{2}{T} \int_0^T F(t) \cos n\omega t dt$$

$$b_n = \frac{2}{T} \int_0^T F(t) \sin n\omega t dt$$

### d) Response to generic forces: Green's functions

• Suppose we want to solve:

$$\underbrace{\left( \frac{d^2}{dt^2} + 2\beta \frac{d}{dt} + \omega_0^2 \right)}_{\hat{L}} x = F(t) / m$$

← "generic", not necessarily periodic

Green's function method: solve equation for a pulse:

$$F(t) = m \delta(t) : \text{Dirac function: } \int dt \delta(t) f(t) = f(0)$$

" $\delta(t) = 0$  for  $t \neq 0$ "  
" $\delta(0) = \infty$ "

$$\hat{L} G = \delta(t) \quad (G = \text{Green's function of } \hat{L})$$

Suppose we know how to compute  $G$ , then the solution for a general function  $f$  is given by the principle of superposition:

$$F(t) = \int_{-\infty}^{+\infty} dt' \delta(t-t') F(t')$$

← "Sum" =  $\sum_n$       ↳ " $\alpha_n$ "

solution for RHS =  $\delta(t-t')$

$$\text{So } x(t) = \int_{-\infty}^{+\infty} dt' G(t-t') \frac{F(t')}{m}$$

Check:  $\hat{L}x(t) = \int_{-\infty}^{+\infty} dt' \underbrace{\hat{L}G(t-t')}_{\delta(t-t')} \frac{F(t')}{m}$   
 $= \frac{F(t)}{m}$  ✓

How do we compute  $G(t)$ ?

Fourier transform:  $\hat{G}(\omega) = \int_{-\infty}^{+\infty} dt e^{i\omega t} G(t)$   
 $G(t) = \int_{-\infty}^{+\infty} \frac{d\omega}{2\pi} e^{-i\omega t} \hat{G}(\omega)$

note that  $\hat{\delta}(\omega) = \int_{-\infty}^{+\infty} dt e^{i\omega t} \delta(t) = 1$

so in Fourier space:  $(-\omega^2 - 2\beta i\omega + \omega_0^2) \hat{G}(\omega) = 1$

$\hookrightarrow \frac{d}{dt} \rightarrow -i\omega$

$$\Rightarrow \hat{G}(\omega) = -\frac{1}{\omega^2 - \omega_0^2 + 2\beta i\omega} = -\frac{1}{(\omega - \omega_+) (\omega - \omega_-)}$$

$$\omega_{\pm} = -i\beta \pm \sqrt{\omega_0^2 - \gamma^2}$$

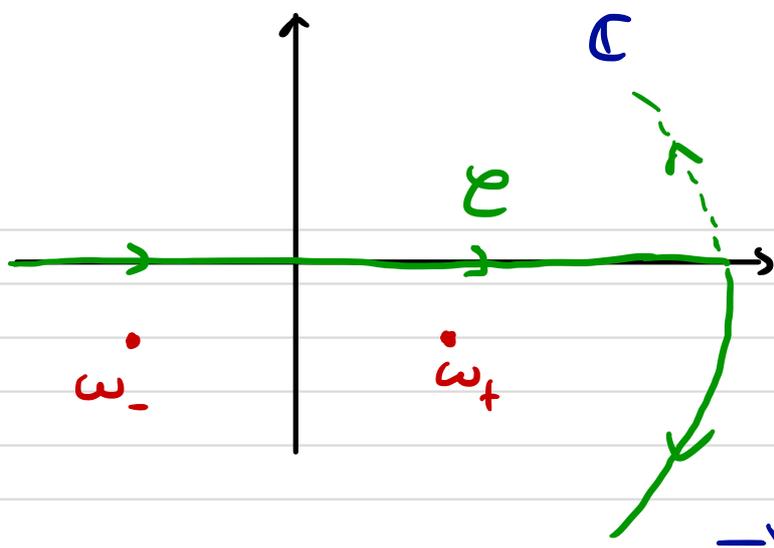
(same frequencies as damped oscillator!)

To get  $G(t)$ , we need to Fourier transform back:

$$G(t) = -\int_{-\infty}^{+\infty} \frac{d\omega}{2\pi} \frac{e^{-i\omega t}}{(\omega - \omega_+) (\omega - \omega_-)}$$

take  $\gamma < \omega_0$   
for simplicity  
(underdamped)

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if  $t < 0$ , close contour in upper half plane

$$\omega = \omega_R + i\omega_I$$

$$-i\omega t = -i\omega_R t + \underbrace{\omega_I t}_{< 0}$$

$\Rightarrow G(t) = 0$  if  $t < 0$   
(no pole)

if  $t > 0$ , use residues:  $G(t) = -\frac{2\pi i}{2\pi} \left[ \frac{e^{-i\omega_+ t}}{\omega_+ - \omega_-} + \frac{e^{-i\omega_- t}}{\omega_- - \omega_+} \right]$

step function:  $\Theta(t) = 1$  if  $t \geq 0$   
 $= 0$  otherwise

$\Rightarrow G(t) = i\Theta(t) \times$

$$e^{-\beta t} \times \frac{1}{2\sqrt{\omega_0^2 - \beta^2}} \left[ -e^{-i\sqrt{\omega_0^2 - \beta^2} t} + e^{i\sqrt{\omega_0^2 - \beta^2} t} \right]$$

$$\Rightarrow G(t) = \Theta(t) e^{-\beta t} \frac{\sin\left[\sqrt{\omega_0^2 - \beta^2} t\right]}{\sqrt{\omega_0^2 - \beta^2}}$$

Note:  $G(t-t') = 0$  if  $t < t'$  by causality

$$x(t) = \int_{-\infty}^{+0} dt' G(t-t') F(t')$$

can only depend on  $F(t')$  for  $t' < t$   
("past")  
not on  $F(t')$  for  $t' > t$   
("future")

## II Coupled Oscillations

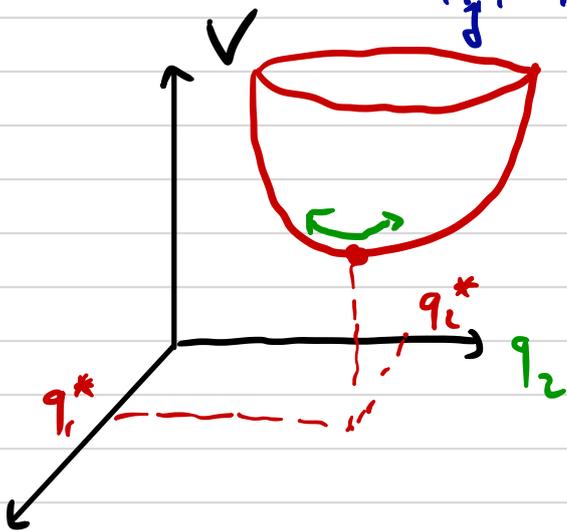
The Lagrangian formalism is very useful to study small oscillations about equilibrium points of systems with many particles / coordinates.

### a) General Approach

$$\mathcal{L} = T(q_i, \dot{q}_i) - V(q_i)$$

$i = 1, \dots, N$   
(generalized coordinates)

Equilibrium:  $\frac{\partial V}{\partial q_i} \Big|_{\{q_j\} = \{q_j^*\}} = 0$  for  $i = 1, \dots, N$



Small oscillations:

$$q_i = q_i^* + \delta q_i$$

Taylor expand:

$$V = V(\{q_i^*\}) + \sum_j \frac{\partial V}{\partial q_j} \Big|_{q=q^*} \delta q_j + \frac{1}{2} \sum_{j,k} \frac{\partial^2 V}{\partial q_j \partial q_k} \Big|_{q^*} \delta q_j \delta q_k + \dots$$

In general,  $T$  is quadratic in  $\dot{q}_j$ :  $T = \sum_{ij} \frac{1}{2} T_{ij} \dot{\delta q}_i \dot{\delta q}_j$

where  $T_{ij} = \frac{\partial^2 T}{\partial \dot{q}_i \partial \dot{q}_j} \Big|_{\substack{\dot{q}_k=0 \\ q_k=q_k^*}}$

$$\Rightarrow L \approx \frac{1}{2} \sum_{ij} T_{ij} \delta \dot{q}_i \delta \dot{q}_j - \frac{1}{2} \sum_{ij} V_{ij} \delta q_i \delta q_j + \dots$$

neglect  $\frac{\partial^2 V}{\partial \dot{q}_i \partial \dot{q}_j} \Big|_{q_k^*}$

$\hat{V} = N \times N$  matrix with matrix elements  $V_{ij}$

should be **positive definite** = all eigenvalues should be positive

in order for the equilibrium point to be stable (local minimum vs maximum or saddle point)

Equations of motion:  $\frac{d}{dt} \frac{\partial L}{\partial \dot{q}_i} = \frac{\partial L}{\partial q_i}$

$$\Leftrightarrow \sum_j T_{ij} \delta \ddot{q}_j = - \sum_j V_{ij} \delta q_j \quad i=1, \dots, N$$

$$\Leftrightarrow \hat{T} \delta \ddot{q} + \hat{V} \delta q = \vec{0}$$

$$\delta \vec{q} = \begin{pmatrix} q_1 \\ \vdots \\ q_N \end{pmatrix}$$

Matrix notation

→ Coupled harmonic oscillators

Look for solutions:  $\delta \vec{q}(t) = \vec{s} e^{i\omega t}$

$$\Rightarrow \hat{T} (i\omega)^2 \vec{s} + \hat{V} \vec{s} = \vec{0}$$

$$\Leftrightarrow (\hat{V} - \omega^2 \hat{T}) \vec{s} = \vec{0} \quad \text{"Eigensystem"}$$

or in components:  $\sum_j (V_{ij} - \omega^2 T_{ij}) s_j = 0 \quad \forall i$

\* In many cases, we'll have  $\hat{T}_{ij} = m_i \delta_{ij}$  and the above problem reduces to diagonalizing  $\hat{V}$ .

\* Non-trivial solution:

$$\text{Det}(\hat{V} - \omega^2 \hat{T}) = 0$$

$N$  solutions:  
 $\rightarrow \omega_1^2, \dots, \omega_N^2$   
 Characteristic Equation

(all solutions are real and positive if  $q^*$  is a real minimum)

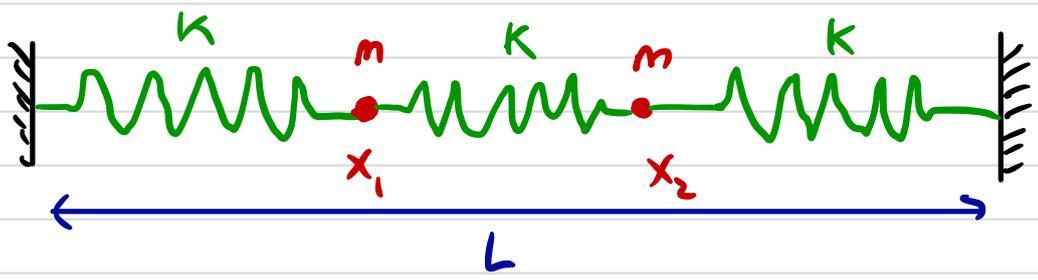
For every  $\omega_k$ ,  $\vec{s}_k$ : eigenvectors  $\Rightarrow$  "Normal Modes"

General solution:  $\delta \vec{q} = \sum_k \alpha_k \vec{s}_k e^{i\omega_k t} + \beta_k \vec{s}_k e^{-i\omega_k t}$   
 (linear combination)  
 $\uparrow$  complex constants, fixed by initial conditions.

$\Rightarrow$  general "recipe" to study coupled oscillations

Q) Examples

①



length at rest:  $L_0 = L/3$

$$V = \frac{1}{2} k (x_1 - L_0)^2 + \frac{1}{2} k (x_2 - x_1 - L_0)^2 + \frac{1}{2} k (L - x_2 - L_0)^2$$

Equilibrium:  $\frac{\partial V}{\partial x_i} = 0 \Rightarrow x_1 = L_0, x_2 = 2L_0$

small variations:  $V = \frac{1}{2} k (\delta x_1)^2 + \frac{1}{2} k (\delta x_2 - \delta x_1)^2 + \frac{1}{2} k (\delta x_2)^2$

$\delta x_1^2 + \delta x_2^2 - 2\delta x_1 \delta x_2$

$$L = \frac{1}{2} m (\dot{\delta x}_1^2 + \dot{\delta x}_2^2) - \frac{1}{2} k \left[ \delta x_1^2 + \delta x_2^2 + \underbrace{(\delta x_1 - \delta x_2)^2}_{\text{coupling}} \right]$$

$$\hat{T} = \begin{pmatrix} m & 0 \\ 0 & m \end{pmatrix} \quad \hat{V} = k \begin{pmatrix} 2 & -1 \\ -1 & 2 \end{pmatrix}$$

Characteristic equation:  $\begin{vmatrix} 2k - \omega^2 m & -k \\ -k & 2k - \omega^2 m \end{vmatrix} = 0$

$$\Leftrightarrow (2k - \omega^2 m)^2 = k^2 \Leftrightarrow 2k - \omega^2 m = \pm k$$

$$\Leftrightarrow \omega_+^2 = \frac{3k}{m}, \quad \omega_-^2 = \frac{k}{m}$$

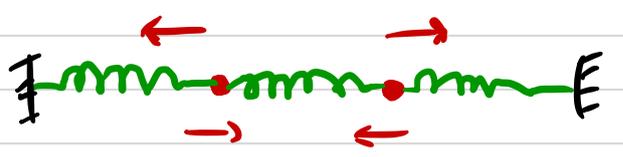
Normal modes:  $(\hat{V} - \omega_{\pm}^2 \hat{T}) \begin{pmatrix} S_1^{\pm} \\ S_2^{\pm} \end{pmatrix} = 0$

$$\begin{pmatrix} 2k - \frac{3k}{m}m & -k \\ -k & 2k - \frac{3k}{m}m \end{pmatrix} \begin{pmatrix} S_1^+ \\ S_2^+ \end{pmatrix} = 0$$

$$\Leftrightarrow \begin{pmatrix} -1 & -1 \\ -1 & -1 \end{pmatrix} \begin{pmatrix} S_1^+ \\ S_2^+ \end{pmatrix} = 0 \Leftrightarrow S_1^+ = -S_2^+$$

$$\Leftrightarrow \vec{S}^+ = \begin{pmatrix} 1/\sqrt{2} \\ -1/\sqrt{2} \end{pmatrix}$$

The motion with frequency  $\omega_+$  corresponds to  $\delta x_1(t) = -\delta x_2(t)$



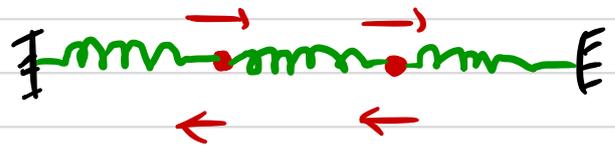
Oscillations out of phase in opposite directions.

$$\begin{pmatrix} 2k - \frac{k}{m}m & -k \\ -k & 2k - \frac{k}{m}m \end{pmatrix} \begin{pmatrix} S_1^- \\ S_2^- \end{pmatrix} = 0$$

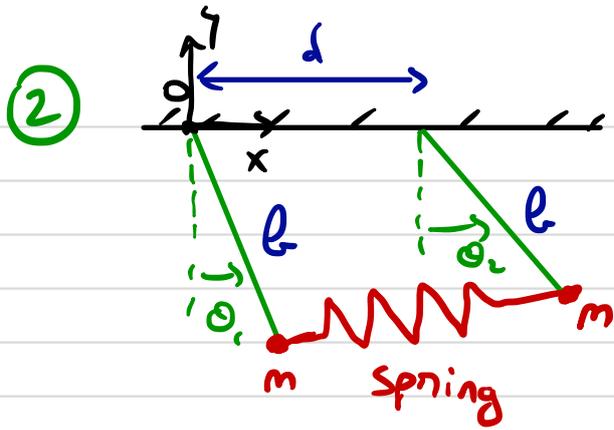
$$\Leftrightarrow \begin{pmatrix} 1 & -1 \\ -1 & 1 \end{pmatrix} \begin{pmatrix} S_1^- \\ S_2^- \end{pmatrix} = 0 \Rightarrow S_1^- = S_2^-$$

$$\Leftrightarrow \vec{S}^- = \begin{pmatrix} 1/\sqrt{2} \\ 1/\sqrt{2} \end{pmatrix}$$

The motion with frequency  $\omega_-$  corresponds to  $\delta x_1 = +\delta x_2$ :



same direction.



Assume that spring is unstretched in equilibrium position  $\theta_1 = \theta_2 = 0$

$$T = \frac{1}{2} m l^2 \dot{\theta}_1^2 + \frac{1}{2} m l^2 \dot{\theta}_2^2$$

$$V = \underbrace{-mg l \cos \theta_1 - mg l \cos \theta_2}_{\text{gravity}} + \frac{1}{2} k (l - l_0)^2$$

$$(x_1, y_1) = (l \sin \theta_1, -l \cos \theta_1)$$

$$(x_2, y_2) = (d + l \sin \theta_2, -l \cos \theta_2)$$

$$l = \sqrt{(x_1 - x_2)^2 + (y_1 - y_2)^2} = \sqrt{(d + l \sin \theta_2 - l \sin \theta_1)^2 + l^2 (\cos \theta_1 - \cos \theta_2)^2}$$

$$l_0 = d$$

Small angles approximations:  $\sin \theta \approx \theta$   
 $\cos \theta \approx 1 - \theta^2/2$

$$l \approx \sqrt{(d + l(\theta_2 - \theta_1))^2 + l^2 \left( \frac{\theta_2^2}{2} - \frac{\theta_1^2}{2} \right)^2}$$

order  $\theta^4$ : drop

$$\approx d + l(\theta_2 - \theta_1)$$

$$\Rightarrow (l - l_0)^2 = l^2 (\theta_2 - \theta_1)^2$$

$$\Rightarrow V = \frac{mg l}{2} (\theta_1^2 + \theta_2^2) + \frac{1}{2} k l^2 (\theta_2 - \theta_1)^2 + \text{Constant}$$

$$\hat{T} = \begin{pmatrix} ml^2 & 0 \\ 0 & ml^2 \end{pmatrix}$$

$$\hat{V} = \begin{pmatrix} mgl + kl^2 & -kl^2 \\ -kl^2 & mgl + kl^2 \end{pmatrix}$$

$$\det(\hat{V} - \hat{T}\omega^2) = \begin{vmatrix} mgl + kl^2 - \omega^2 ml^2 & -kl^2 \\ -kl^2 & mgl + kl^2 - \omega^2 ml^2 \end{vmatrix}$$

$$= 0 \Rightarrow mgl + kl^2 - \omega^2 ml^2 = \pm kl^2$$

+ sign:  $\omega_+^2 ml^2 = mgl \Rightarrow \omega_+ = \sqrt{g/l}$

- sign:  $mgl + 2kl^2 = \omega_-^2 ml^2$

$$\Rightarrow \omega_- = \sqrt{\frac{g}{l} + \frac{2k}{m}}$$