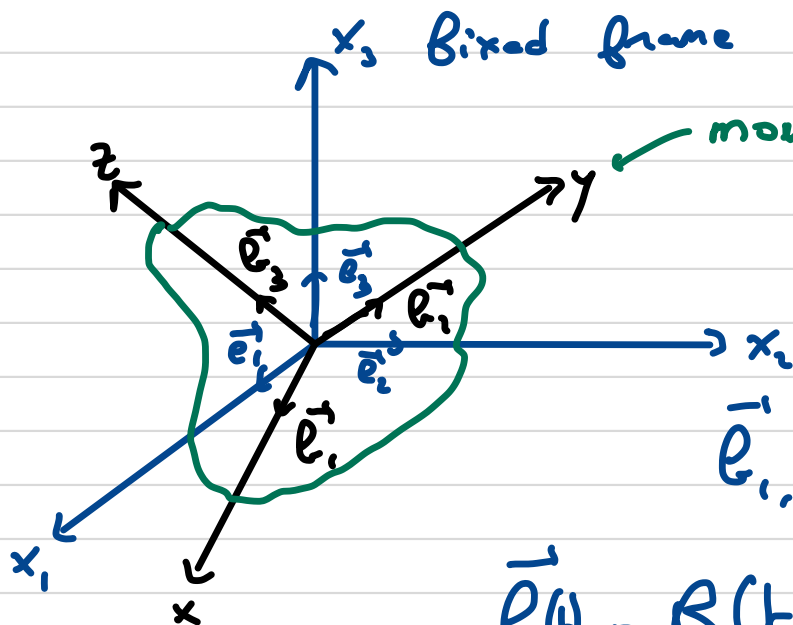


Rigid Bodies

① Kinematics : Rigid Bodies : N points s.t. $\|\vec{r}_\alpha - \vec{r}_\beta\| = \text{constant}$

$$M = \sum_{\alpha} m_{\alpha} = \int d^3\vec{r} \rho(\vec{r}) \quad \text{with} \quad \rho(\vec{r}) = \sum_{\alpha} m_{\alpha} \delta(\vec{x} - \vec{x}_{\alpha})$$

6 degrees of freedom: Translations + Rotations
3 3



$$\vec{e}_i \cdot \vec{e}_j = R_i^T \cdot \vec{e}_j = \delta_{ij}$$

$\vec{e}_1, \vec{e}_2, \vec{e}_3$ all depend on time

$$\vec{e}_i(t) = R_{ij}(t) \vec{e}_j \quad \text{with} \quad R(t) \in SO(3)$$

$$R_{ij}(t) = \vec{e}_i(t) \cdot \vec{e}_j$$

and $\vec{e}_i = (R^T)_{ij} \vec{e}_j = (\vec{e}_i \cdot \vec{e}_j) \vec{e}_j$

$$\vec{r}(t) = \underbrace{x_i(t)}_{\substack{\uparrow \\ \text{point in body}}} \underbrace{\vec{e}_i}_{\substack{\uparrow \\ \text{fixed frame}}} = \underbrace{x_i}_{\substack{\uparrow \\ \text{body frame}}} \vec{e}_i(t) \quad \text{with}$$

$$\vec{x} = R \vec{x}(t)$$

$$\vec{x}(t) = R^T \vec{x}$$

We have $\frac{d\vec{n}}{dt} = \dot{x}_i(t)\vec{e}_i = x_i \dot{\vec{e}}_i = x_i R_{ij} \dot{\vec{e}}_j$

The body frame is changing with time:

$$\dot{\vec{e}}_i = R_{ij} \dot{\vec{e}}_j = R_{ij} \underbrace{(R^{-1})}_{(R^T)_{jk}} \dot{\vec{e}}_k \equiv \omega_{ik} \vec{e}_k$$

$$\omega = \dot{R} R^T \Leftrightarrow \boxed{\dot{R} = \omega R} \quad R = \text{Temp} \left(\int_0^t \omega \right)$$

↑
time ordered exponential

$$R R^T = \mathbb{1} \Rightarrow \dot{R} R^T + R \dot{R}^T = 0$$

$$\Leftrightarrow \omega + \omega^T = 0 : \omega \text{ antisymmetric}$$

We can parametrize: $\omega_{ij} = \epsilon_{ijk} \omega_k$ L vector

$$\omega = \begin{pmatrix} 0 & -\omega_3 & \omega_2 \\ \omega_3 & 0 & -\omega_1 \\ -\omega_2 & \omega_1 & 0 \end{pmatrix} : \dot{\vec{e}}_i = \epsilon_{ijk} \omega_k \vec{e}_j = -\omega_k \underbrace{\epsilon_{ikj}}_{\vec{e}_i \times \vec{e}_k} \vec{e}_j = \vec{\omega} \times \vec{e}_i$$

$\vec{\omega} = \omega_k \vec{e}_k$: instantaneous angular velocity.

$$\boxed{\dot{\vec{e}}_i = \vec{\omega} \times \vec{e}_i}$$



so $\dot{\vec{n}} = x_i \dot{\vec{e}}_i = \vec{\omega} \times (x_i \vec{e}_i) = \omega \times \vec{n}$

① Inertia tensor : start with rotating body

$$T = \frac{1}{2} \sum_{\alpha} m_{\alpha} \vec{\pi}_{\alpha}^2 = \frac{1}{2} \sum_{\alpha} m_{\alpha} (\vec{\omega} \times \vec{\pi}_{\alpha}) \cdot (\vec{\omega} \times \vec{\pi}_{\alpha})$$

↑ labels particles R_{α}

$$(\vec{a} \times \vec{b})^2 = \epsilon_{ijk} a_j b_k \epsilon_{ilm} a_l b_m = (\delta_{jl} \delta_{km} - \delta_{jm} \delta_{kl})$$

$$\times a_j b_k a_l b_m$$

$$= a^2 b^2 - (\vec{a} \cdot \vec{b})^2$$

$$\Rightarrow T = \frac{1}{2} \sum_{\alpha} m_{\alpha} \left[\omega^2 \vec{\pi}_{\alpha}^2 - (\vec{\pi}_{\alpha} \cdot \vec{\omega})^2 \right]$$

$$= \frac{1}{2} \sum_{\alpha} m_{\alpha} \omega_i \left(\delta_{ij} \vec{\pi}_{\alpha}^2 - x_i^{\alpha} x_j^{\alpha} \right) \omega_j$$

$$= \frac{1}{2} \omega_i I_{ij} \omega_j \quad \text{with :}$$

$$I_{ij} = \sum_{\alpha} m_{\alpha} \left(\delta_{ij} \vec{\pi}_{\alpha}^2 - x_i^{\alpha} x_j^{\alpha} \right)$$

Inertia
Tensor

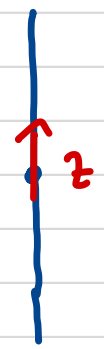
$$I = \int d\vec{\pi} \rho(\vec{\pi}) \begin{pmatrix} y^2 + z^2 & -xy & -xz \\ -xy & x^2 + z^2 & -yz \\ -xz & -yz & x^2 + y^2 \end{pmatrix}$$

defined with respect to a point (origin)

• I is the independent, since coordinates $x_1, x_2, x_3 = x, y, z$ are measured with respect to the body frame.

• I is symmetric: we can diagonalize it using an orthogonal matrix: eigenvalues: **Principle moment of inertia**
 eigenvectors: **Principle axes**

Examples **Rod** (about its center): $\rho = M/\rho$



$$I_z = 0$$

$$I_x = I_y = \int_{-\rho/2}^{\rho/2} x^2 \rho dx = \frac{1}{12} M \rho^2$$

Cylinder: $\rho = M/(\pi r_0^2 h)$



$$I_z = h \int_{r < r_0} dx dy (x^2 + y^2) \rho = 2\pi \int_0^{r_0} dr r^3 \frac{M}{\pi r_0^2}$$

$\underbrace{\int_0^{r_0} dr r^3}_{r_0^4/4}$

$$= \frac{1}{2} M r_0^2$$

$$I_x = I_y = \int dx dy dz \frac{M}{\pi r_0^2 h} (z^2 + y^2)$$

$$= \frac{M}{h} \int_{-h/2}^{h/2} dz z^2 + \frac{M}{\pi r_0^2} \int dx dy y^2 = \frac{1}{2} M h^2 + \frac{1}{4} M r_0^2$$

$\underbrace{\int_0^{2\pi} d\theta \int_0^{r_0} dr r^3 \sin^2 \theta}_{\pi r_0^4/4} = \pi r_0^4/4$

Parallel axis theorem : $I_{\vec{y}}$ = new origin displaced by \vec{y} from center of mass

$$(I_{\vec{y}})_{ij} = (I_{com})_{ij} + M(\vec{y}^2 \delta_{ij} - y_i y_j)$$

Proof : $(I_{\vec{y}})_{ij} = \sum_{\alpha} m_{\alpha} \{ (\vec{\pi}_{\alpha} - \vec{y})^2 \delta_{ij} - (\pi_{\alpha} - y)_i (\pi_{\alpha} - y)_j \}$

$$= \underbrace{\sum_{\alpha} m_{\alpha} (\pi_{\alpha}^2 \delta_{ij} - x_i^{\alpha} x_j^{\alpha})}_{(I_{com})_{ij}} + \sum_{\alpha} m_{\alpha} (\vec{y}^2 \delta_{ij} - y_i y_j)$$

$(I_{com})_{ij}$

$$+ \underbrace{\sum_{\alpha} m_{\alpha} (-2\vec{y} \cdot \vec{\pi}_{\alpha} \delta_{ij} + x_i^{\alpha} y_j + y_i x_j^{\alpha})}_{\text{terms that cancel out}}$$

o Since $\sum_{\alpha} m_{\alpha} \vec{\pi}_{\alpha} = 0$
if $0 = \text{center of mass}$

Ex: Inertia tensor of a rod about one of its ends:

$$I_x = \frac{1}{12} M P^2 + M \left(\frac{P}{2}\right)^2 = \frac{1}{3} M P^2$$

Energy : $\vec{\pi}_{\alpha}(t) = \vec{R} + \vec{\delta}_{\alpha}(t)$ with com motion \vec{R}

$$T = \frac{1}{2} \sum_{\alpha} m_{\alpha} \left[\vec{R}^{\dot{2}} + 2\vec{R} \cdot \vec{\delta}_{\alpha}^{\dot{}} + \vec{\delta}_{\alpha}^{\dot{2}} \right]$$

$$T = \frac{1}{2} M \vec{R}^{\dot{2}} + \frac{1}{2} \omega_i I_{ij} \omega_j$$

com motion separates: set $\sum_{\alpha} m_{\alpha} \vec{\delta}_{\alpha}^{\dot{}} = 0$
 $I_{ij} = \sum_{\alpha} m_{\alpha} \delta_{\alpha i} \delta_{\alpha j}$

Angular momentum : $\vec{L} = \sum_{\alpha} m_{\alpha} \vec{r}_{\alpha} \times \dot{\vec{r}}_{\alpha}$

$= \sum_{\alpha} m_{\alpha} \vec{r}_{\alpha} \times (\vec{\omega} \times \vec{r}_{\alpha}) = \sum_{\alpha} m_{\alpha} (\vec{r}_{\alpha}^2 \vec{\omega} - (\vec{\omega} \cdot \vec{r}_{\alpha}) \vec{r}_{\alpha})$

$\Rightarrow \vec{L} = I \vec{\omega}$

time dependent!

In body frame : $\vec{L} = L_i \vec{e}_i$: $L_i = I_{ij} \cdot \omega_j$
 $\vec{\omega} = \omega_i \vec{e}_i$

⚠ \vec{L} and $\vec{\omega}$ point in different directions in general!

III Free motion of a rigid body

Euler's equations : $\vec{F}_{ext} = \vec{0}$, $\vec{R} = \text{fixed} = \text{origin}$

we have $\frac{d\vec{L}}{dt} = \vec{0}$ in body frame: $\dot{L}_i \vec{e}_i + L_i \dot{\vec{e}}_i = \vec{0}$

$\Leftrightarrow L_i \vec{e}_i + L_j \vec{\omega} \times \vec{e}_j = \vec{0}$ with $\vec{L} = I \vec{\omega}$

Let's choose $\vec{e}_i = \text{principal axes}$: $L_i = I_i \omega_i$
i=1,2,3=x,y,z

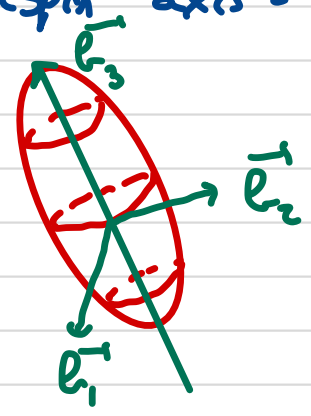
$\Rightarrow I_i \dot{\omega}_i + \epsilon_{ijk} \omega_j I_k \omega_k = 0$

Euler's equations:

$I_1 \dot{\omega}_1 + \omega_2 \omega_3 (I_3 - I_2) = 0$
 $I_2 \dot{\omega}_2 + \omega_1 \omega_3 (I_1 - I_3) = 0$
 $I_3 \dot{\omega}_3 + \omega_1 \omega_2 (I_2 - I_1) = 0$

Free tops : sphere : $\vec{\omega} = \text{constant}$ (spin axis = const)

Symmetric top : $I_1 = I_2 \neq I_3$



$$I_3 \dot{\omega}_3 = 0$$

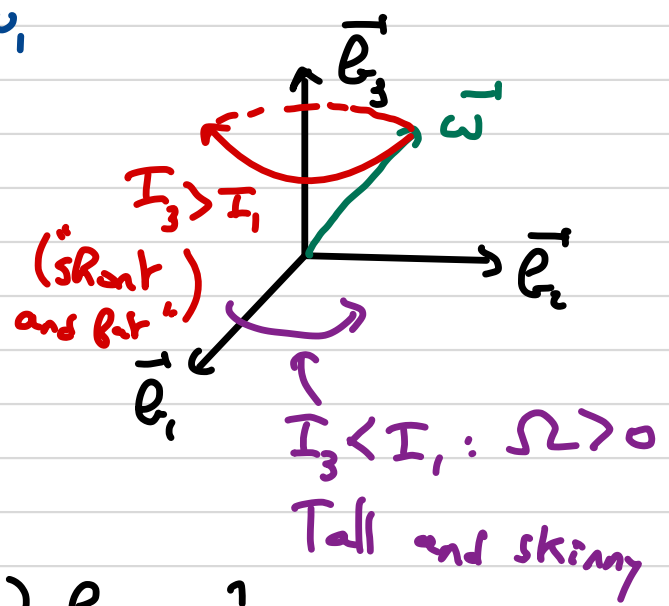
$$\dot{\omega}_1 = \frac{\omega_2 \omega_3}{I_1} (I_1 - I_3)$$

$$\dot{\omega}_2 = -\frac{\omega_1 \omega_3}{I_1} (I_1 - I_3)$$

$$\Rightarrow \omega_3 = \text{const}, \quad \dot{\omega}_1 = \Omega \omega_2 \quad \text{with } \Omega = \omega_3 \left(1 - \frac{I_3}{I_1}\right)$$

$$\dot{\omega}_2 = -\Omega \omega_1$$

we have
$$\begin{cases} \omega_1 = \omega_0 \sin \Omega t \\ \omega_2 = \omega_0 \cos \Omega t \end{cases}$$



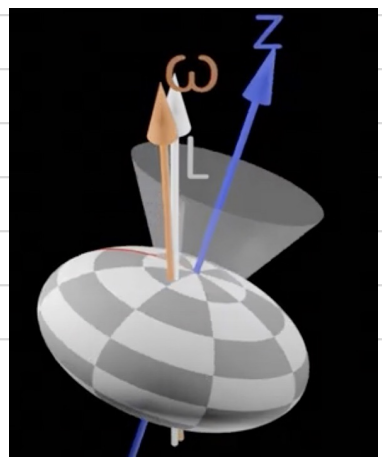
$\vec{\omega}$ precesses about \vec{e}_3 with frequency Ω ! "wobble"

What about in the Lab (fixed) frame?

$$\vec{L} = \text{const},$$

$$\text{and } L_3 = \vec{L} \cdot \vec{e}_3(t) = \text{const}$$

angle between \vec{L} and \vec{e}_3 fixed
see later (Euler's angles)



stability of asymmetric top all I_i 's different.

$\omega_1 = \Omega$, $\omega_2 = \omega_3 = 0$ solves Euler's equation

$$\vec{\omega} = \begin{pmatrix} \Omega + \epsilon_1 \\ \epsilon_2 \\ \epsilon_3 \end{pmatrix} : \begin{cases} I_1 \dot{\epsilon}_1 = 0 \\ I_2 \dot{\epsilon}_2 = \Omega \epsilon_3 (I_3 - I_1) + \mathcal{O}(\epsilon^2) \\ I_3 \dot{\epsilon}_3 = \Omega \epsilon_2 (I_1 - I_2) \end{cases}$$

$$\Rightarrow I_2 \ddot{\epsilon}_2 = \frac{\Omega^2}{I_3} (I_3 - I_1)(I_1 - I_2) \epsilon_2$$

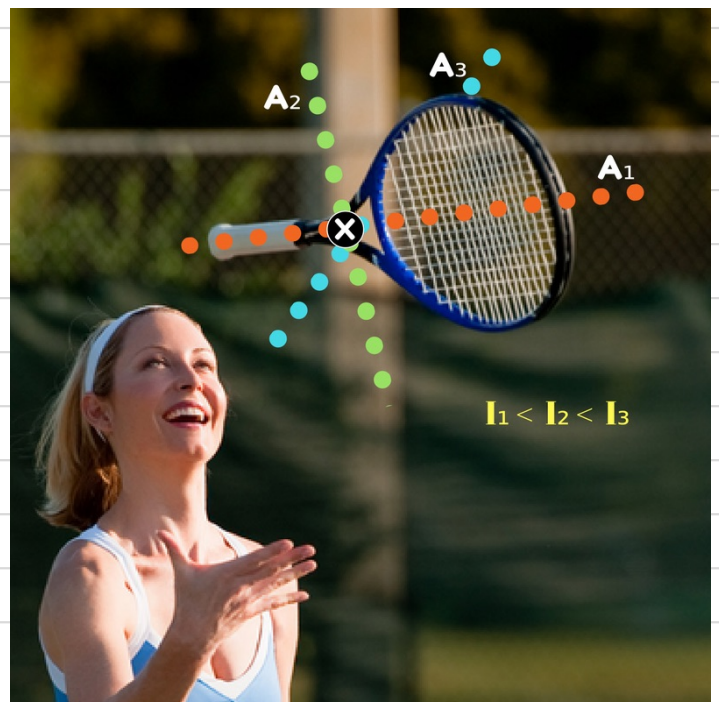
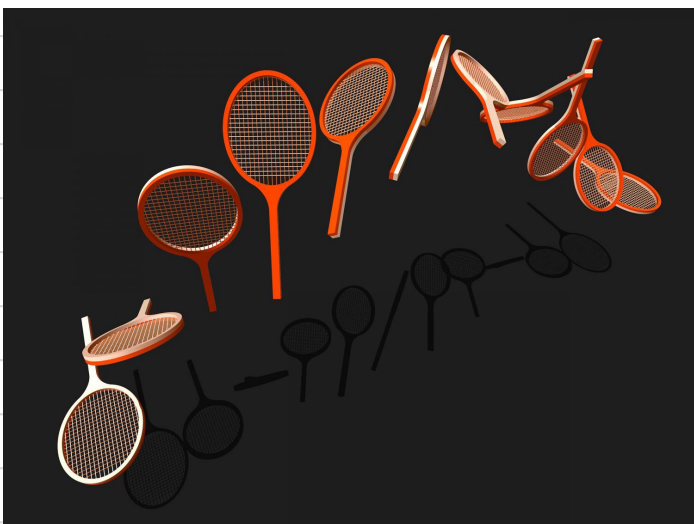
$$\Rightarrow \ddot{\epsilon}_2 + \underbrace{\Omega^2 \left(1 - \frac{I_1}{I_3}\right) \left(1 - \frac{I_1}{I_2}\right)}_{< 0} \epsilon_2 = 0$$

< 0 : unstable : $I_3 < I_1 < I_2$

$I_2 < I_1 < I_3$

Stable if I_1 largest or smallest

"Tennis Racket Theorem"
"unstable axis theorem"



Geometric construction : as before, work in body frame
 \vec{L} constant, but $L_i = I_i \omega_i$ evolve

$$L^2 = I_1^2 \omega_1^2 + I_2^2 \omega_2^2 + I_3^2 \omega_3^2 \quad \text{conserved}$$

$$2E = I_1 \omega_1^2 + I_2 \omega_2^2 + I_3 \omega_3^2$$

work with $\alpha_i = L_i / L$:

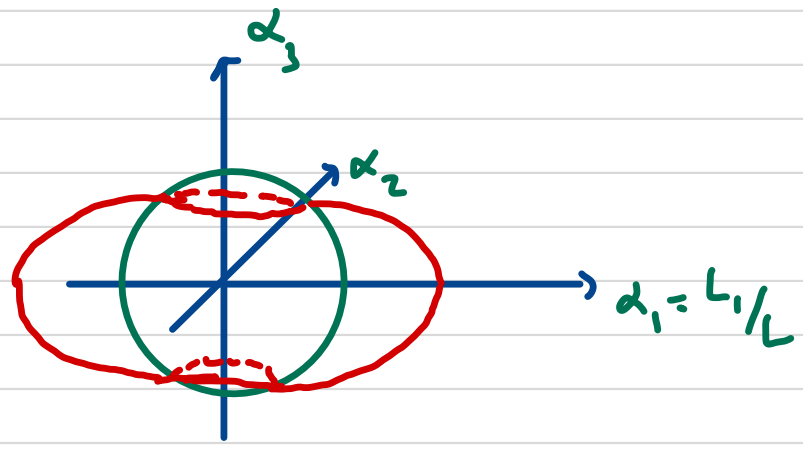
$$\left\{ \begin{aligned} \alpha_1^2 + \alpha_2^2 + \alpha_3^2 &= 1 \\ \frac{\alpha_1^2}{I_1} + \frac{\alpha_2^2}{I_2} + \frac{\alpha_3^2}{I_3} &= \frac{2E}{L^2} \end{aligned} \right.$$

sphere

ellipsoid : $I_3 > I_2 > I_1 > 0$

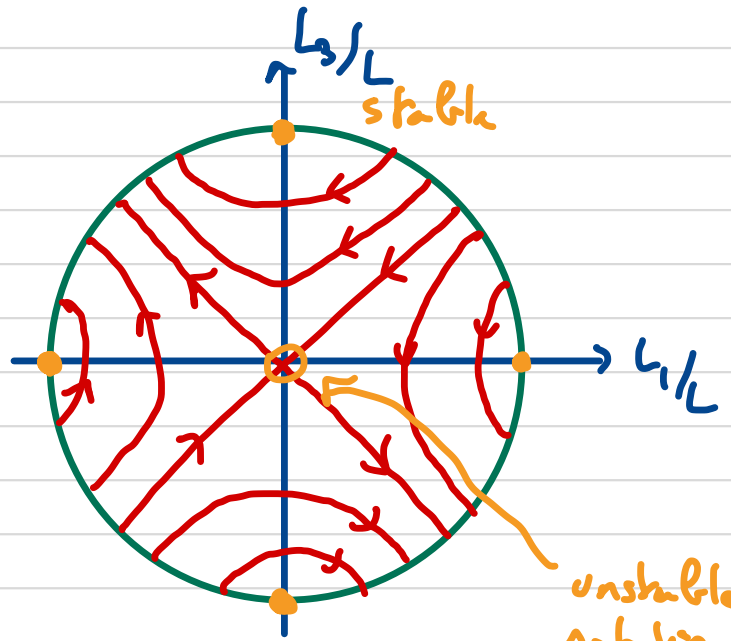
x_1 : major axis

x_3 : minor axis



$\frac{1}{I_3} < \frac{2E}{L^2} < \frac{1}{I_2}$: precession around \vec{e}_3

$\frac{1}{I_2} < \frac{2E}{L^2} < \frac{1}{I_3}$: precession around \vec{e}_1



$$\frac{2E}{L^2} = \frac{1}{I_2}$$

$$\Rightarrow L_3 = \pm \sqrt{\frac{(I_2/I_1 - 1)}{(1 - I_2/I_3)}}$$

planes intersecting sphere : circles sharing points $(0, \pm 1, 0)$

unstable uniform rotation about \vec{e}_2

Rotation is **periodic** in Body frame (but not in fixed frame!!)

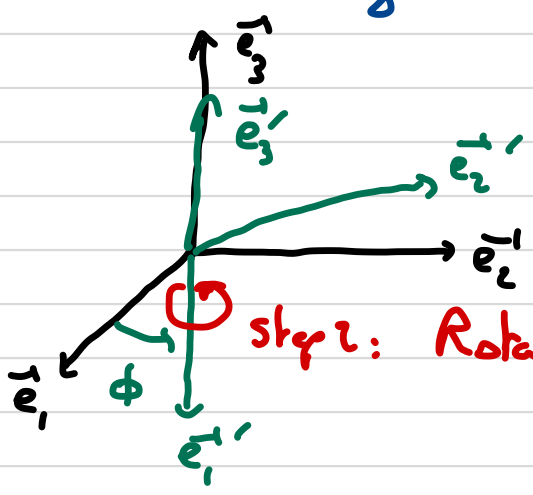
In fixed frame: $T = \frac{1}{\omega} \vec{L} \cdot \vec{\omega}$: $\vec{\omega}$ has constant component along \vec{L}

IV Euler's angles : explicit parametrization of $R(t)$

$$\vec{e}_i(t) = R_{ij}(t) \vec{e}_j$$

$$\{ \vec{e}_i \} \xrightarrow{R_3(\phi)} \{ \vec{e}_i' \} \xrightarrow{R_1(\theta)} \{ \vec{e}_i'' \} \xrightarrow{R_3(\psi)} \{ \vec{e}_i \}$$

① $\vec{e}_i' = [R_3(\phi)]_{ij} \vec{e}_j$ with $R_3(\phi) = \begin{pmatrix} \cos\phi & \sin\phi & 0 \\ -\sin\phi & \cos\phi & 0 \\ 0 & 0 & 1 \end{pmatrix}$



② $R_1(\theta) = \begin{pmatrix} 1 & 0 & 0 \\ 0 & \cos\theta & \sin\theta \\ 0 & -\sin\theta & \cos\theta \end{pmatrix}$

$$\vec{e}_i'' = [R_1(\theta)]_{ij} \vec{e}_j'$$

step 2: Rotate around \vec{e}_3'

③ Rotate around \vec{e}_3'' : $\vec{e}_i = [R_3(\psi)]_{ij} \vec{e}_j''$

$$R(t) = R_3(\psi) R_1(\theta) R_3(\phi)$$

and: $\vec{\omega} = \dot{\phi} \vec{e}_3 + \dot{\theta} \vec{e}_1' + \dot{\psi} \vec{e}_3$

expand in body frame: $\vec{e}_3 = \sin\theta \sin\psi \vec{e}_1 + \sin\theta \cos\psi \vec{e}_2 + \cos\theta \vec{e}_3'$
 $\vec{e}_1' = \cos\psi \vec{e}_1 - \sin\psi \vec{e}_2$

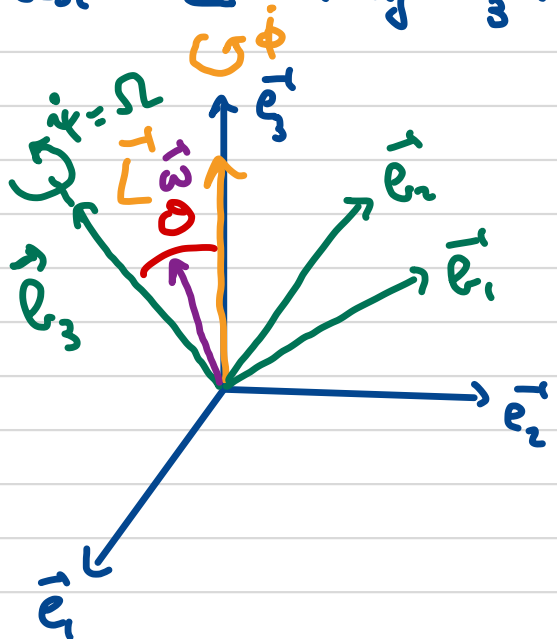
$$\Rightarrow \vec{\omega} = \begin{pmatrix} \dot{\phi} \sin\theta \sin\psi + \dot{\theta} \cos\psi \\ \dot{\phi} \sin\theta \cos\psi - \dot{\theta} \sin\psi \\ \dot{\psi} + \dot{\phi} \cos\theta \end{pmatrix}$$

(in body frame basis)

E_x : free top: $(\omega_1, \omega_2) = \omega_0 (\sin\Omega t, \cos\Omega t)$

$\Omega = \omega_3 (1 - I_3/I_1)$ and constant ω_3 spin

Choose \vec{L} along \vec{e}_3 : $L_3 = I_3 \omega_3$ conserved



angle between \vec{e}_3 and \vec{e}_3'
 = constant = θ

$\dot{\theta} = 0$

$\omega_3 = \dot{\psi} + \dot{\phi} \cos\theta = \text{constant}$

$\omega_1 = \dot{\phi} \sin\theta \sin\psi = \omega_0 \sin\Omega t$

$\omega_2 = \dot{\phi} \sin\theta \cos\psi = \omega_0 \cos\Omega t$

$\Rightarrow \psi = \Omega t$ and $\Omega + \dot{\phi} \cos\theta = \omega_3$

$\Rightarrow \dot{\phi} = \frac{I_3 \omega_3}{I_1 \cos\theta}$

wobble rate = precession
 $\dot{\psi} = \text{spin}$

Heavy symmetric top: add gravity, $I_1 = I_2$
Fixed point $P (\neq \text{COM})$

$$L = \frac{1}{2} \tilde{I}_1 (\omega_1^2 + \omega_2^2) + \frac{1}{2} I_3 \omega_3^2 - Mgl \cos \theta$$

$$= \frac{1}{2} \tilde{I}_1 (\dot{\theta}^2 + \sin^2 \theta \dot{\phi}^2) + \frac{1}{2} I_3 (\dot{\psi} + \cos \theta \dot{\phi})^2 - Mgl \cos \theta$$

$\tilde{I}_1 = I_1 + Ml^2$: $l = \text{dist}(P, \text{Center of mass})$

$$\frac{\partial L}{\partial \dot{\psi}} = 0 \Rightarrow p_\psi = (\dot{\psi} + \cos \theta \dot{\phi}) I_3 = I_3 \omega_3 = \text{const.}$$

$$\frac{\partial L}{\partial \dot{\phi}} = 0 \Rightarrow p_\phi = \tilde{I}_1 \sin^2 \theta \dot{\phi} + \cos \theta p_\psi = \text{const.}$$

$$E \text{ conserved: } E = \frac{1}{2} \tilde{I}_1 (\dot{\theta}^2 + \sin^2 \theta \dot{\phi}^2) + Mgl \cos \theta + \underbrace{\frac{1}{2} I_3 \omega_3^2}_{\text{const}}$$

we have:
$$\dot{\phi} = \frac{p_\phi - \cos \theta p_\psi}{\tilde{I}_1 \sin^2 \theta}$$

$$E = \frac{1}{2} \tilde{I}_1 \dot{\theta}^2 + Mgl \cos \theta + \frac{(p_\phi - \cos \theta p_\psi)^2}{2 \tilde{I}_1 \sin^2 \theta} + \text{const}$$

$V_{\text{eff}}(\theta)$